



A solid tungsten divertor for ASDEX Upgrade

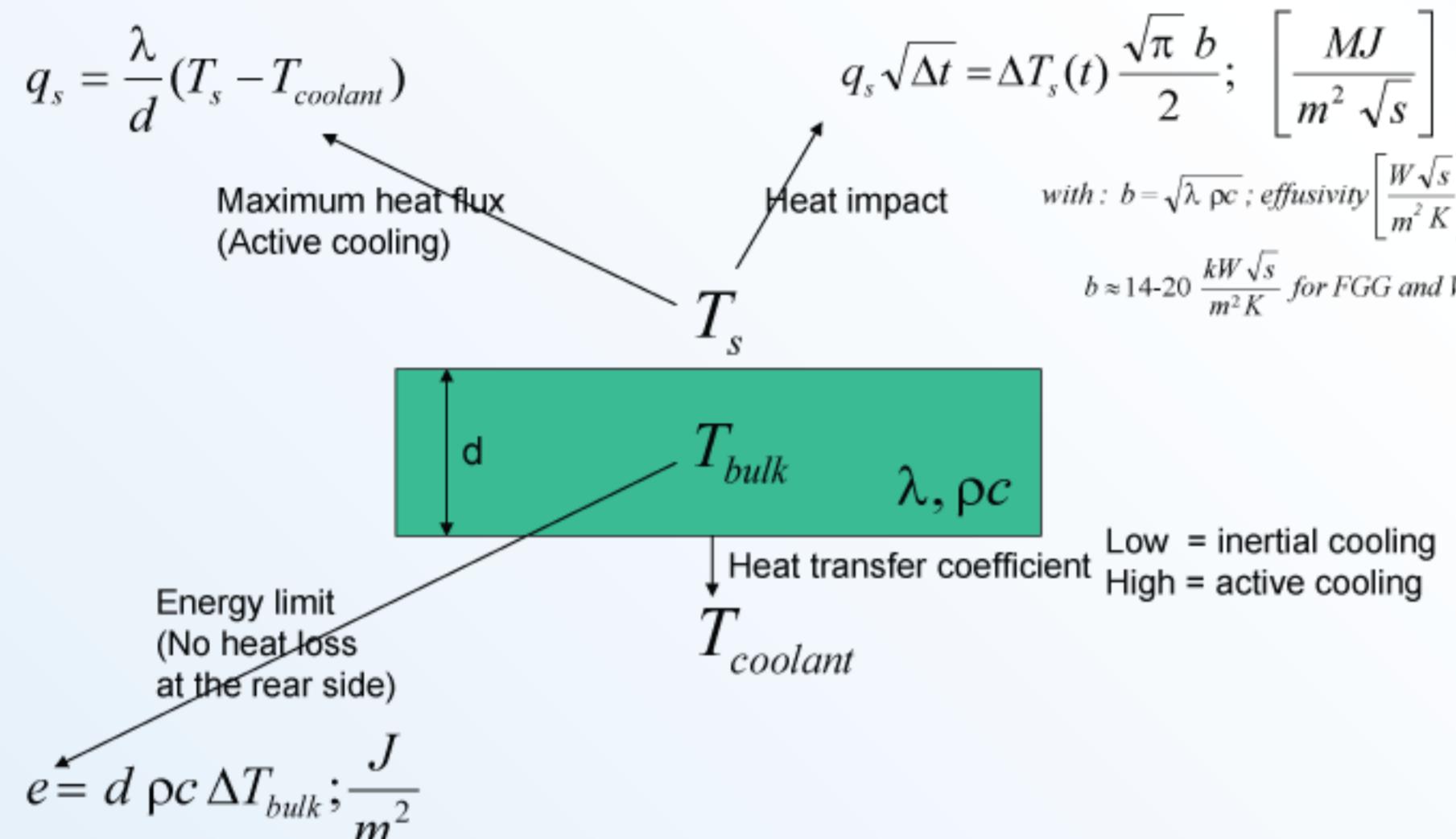
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IPP

Motivation

- ASDEX Upgrade was changed stepwise from a full carbon to a full tungsten experiment.
- The full tungsten first wall is in operation since 2007.
- Coating of fine grain graphite with:
 - W-PVD (4 µm, main chamber)
 - W-VPS (200 µm lower outer divertor)
 - W-CMSII (10 µm lower outer divertor, > 2008)
- Coatings have a lower heat receiving capability compared to carbon. This limits the operational parameters of ASDEX Upgrade.
- A solid tungsten divertor
 - restores the operational windows as for a graphite divertor.
 - allows the investigation of erosion and deuterium retention
 - is flexible for testing of different castellations (width, distance ...)
 - and target tilting/shadowing

Divertor limits



Energy limits

Target material	Target data		Maximum equilibrated temperature / °C		Energy to a single target / MJ/tile		Energy to the divertor / MJ		
	0.5 x target mass / g	Heat capacity J / (g K)	Single event	cyclic	Single event	Cyclic	Number of tiles	Single event	Cyclic
Graphite	470	1.50	550	300	0.38	0.20	128	48	26
Tungsten coated graphite	470	1.50	550	300	0.38	0.20	128	48	26
Solid tungsten	2703	0.15	1000	600	0.40	0.24	128	51	30

Heat impact factor

Target material	Target data (@ 500 °C)				Effusivity	Max. surface temperature	Heat impact factor
	Mass density	Specific heat	Heat conductivity	W/mK			
Graphite	12 g/cm³	1 kJ/kg K	W/mK	70	14	2500	31
Tungsten coated graphite	1850	1500	70	14	1400	4800	48
Solid tungsten	1850	1500	70	14	1400	2800	17

The heat impact factor for tungsten coated graphite restricts the discharge duration on high performance plasmas (> 10 MW/m² stationary) to about 2.6 s.

Design criteria and conceptual design

Do we need an actively cooled divertor? - NO

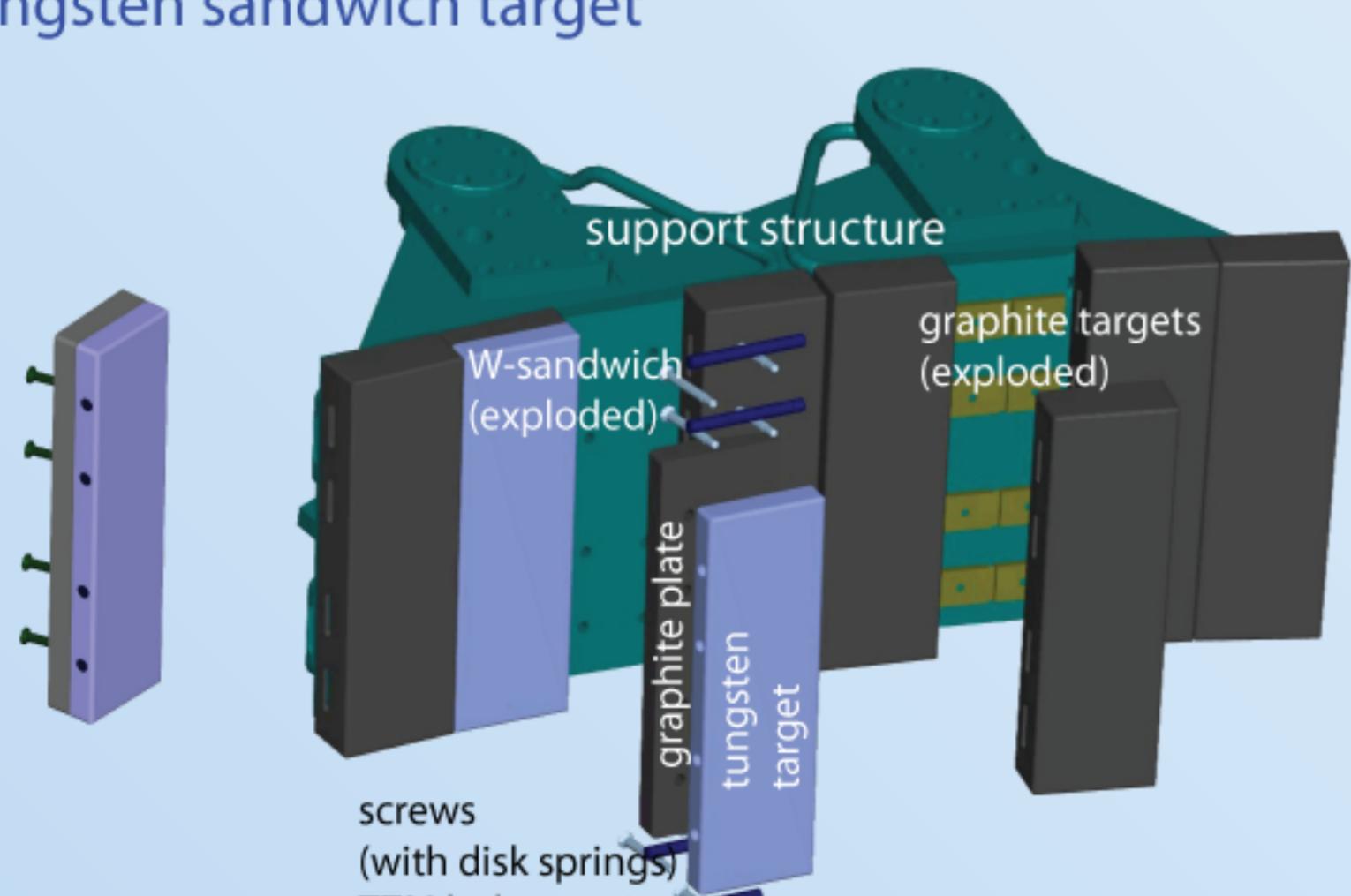
- Max. technical pulse length: 10 s
 - Heating of toroidal field coils
 - Energy consumption from fly wheel generators.
 - DAQ is designed to the technical pulse length.
- Max energy input: 10 s * 28 MW = 280 MJ
- Max tolerable plasma energy input of the present inertially cooled divertor:
 - 78 MJ ~ 8 s with 10 MW - standard operation
 - 144 MJ ~ 5 s with 28 MW - high power shots (single events)
- Problem with the tungsten coated divertor was the surface temperature NOT the energy limit.

Design criteria

The new Div-III should have:

- the same energy receiving capability as a graphite divertor of about 50 MJ
- a tolerable maximum heat load comparable to the uncoated graphite divertor
- compatibility to the existing support structure for the divertor target plates
- an optimized weight that allows handling with a minimum of in vessel tools.
- induced forces that are compatible to the existing support structure.
- a price that fits to the budget
- been realized in the near future

Tungsten sandwich target



- The 30 mm thick tungsten coated graphite tiles are replaced by a tungsten graphite sandwich structure.
- Sandwich: a 15 mm thick solid tungsten plate clamped together with a graphite plate of about 15 mm thickness to the existing cooling structure.
- Target tilting is realized by shaping the graphite plate.
- Flat tungsten plate that to minimize the manufacturing effort and to maximize the flexibility of divertor shaping.
- Increased weight of the divertor plates from about 8 kg/sector to 40 kg/sector. (+20 kg support structure).
- Reduced thickness = reduced heat capacity: this is compensated by using TZM instead of SS = higher tolerable equilibrium temperature of 600 °C and 1000 °C for cyclic load and single events, respectively.
- The energy limit becomes for the solid tungsten sandwich divertor 30 and 50 MJ for cyclic and single loads, respectively.

Forces

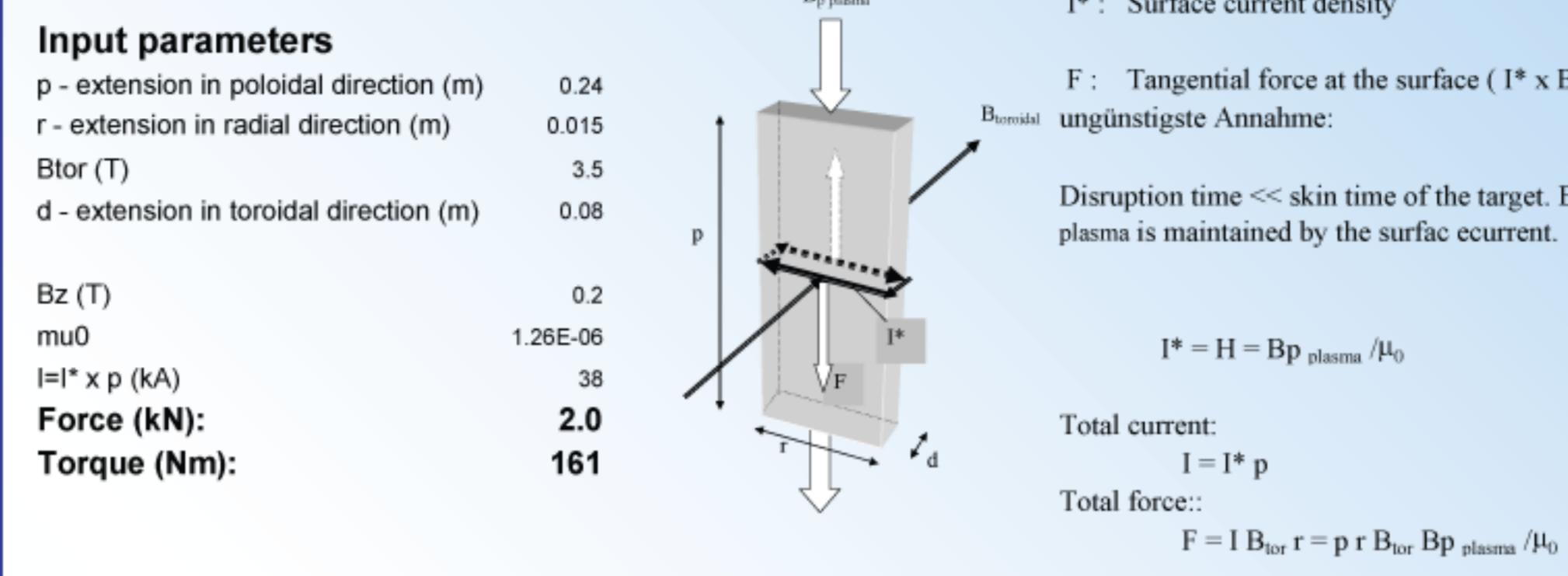
- Divertor II with graphite targets is operated without (major) problems?
- Mechanical loads due to tungsten targets?
 - What is changing due to tungsten targets?
 - Model the divertor structure and calculate mechanical loads.
- Origin of mechanical loads?
 - Eddy currents/Disruptions
 - Worst case estimation (no material dependence)
 - Conventional.
 - Halo currents - NO difference to graphite
- FEM calculations: The divertor structure and the target clamping can withstand these loads.

Halo current

$$F_{halo} = I_{halo} B_{tor} I = 450 kA / 16 \times 3.5 T \times 0.5 m \approx 50 kN$$

Eddy currents: worst case estimation - no material dependence

Force to a divertor tile



Eddy currents: conventional estimation - material dependent

Forces due to eddy currents

Input parameters	poloidal field		radial field		
	Graphite	W	Structure	W	Structure
r, p - extension perp. to dB/dt (m)	0.015	0.015	0.1	0.24	0.24
Btor (T)	3.5	3.5	3.5	3.5	3.5
t - extension in toroidal direction (m)	0.08	0.08	0.4	0.08	0.32
dB/dt (T/s)	100	100	100	10	10
p, r - extension along dB/dt (m)	0.24	0.24	0.015	0.015	0.015
rho - el. Conductivity (Ohm m)	1.0E-05	6.0E-08	7.0E-07	6.0E-08	7.0E-07

Output

Force (kN):	0.01	1.01	1.18	0.17	0.37
Moment (Nm):	0.03	5.71	44.12	15.12	33.18

Modell

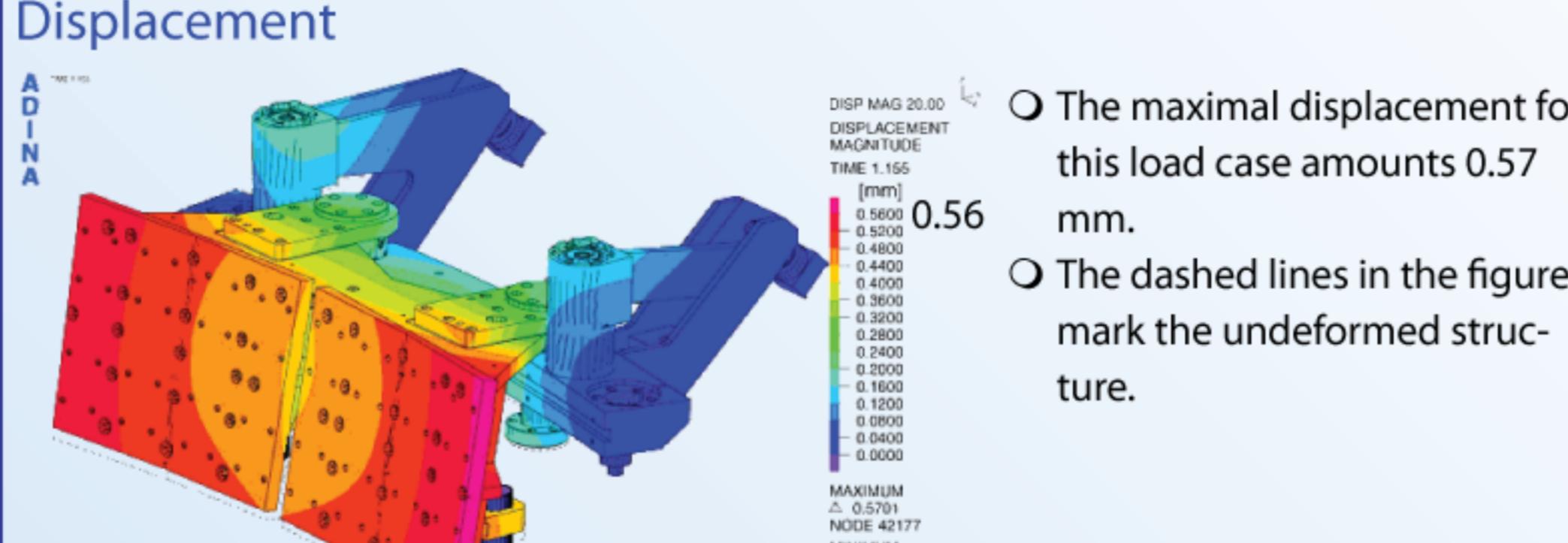
Total force to the half plate

$$F_{tot} = B_t \cdot \dot{B}_p \cdot \frac{p}{\rho} \cdot \frac{t^3}{(1+r^2/t^2)^{24}}$$

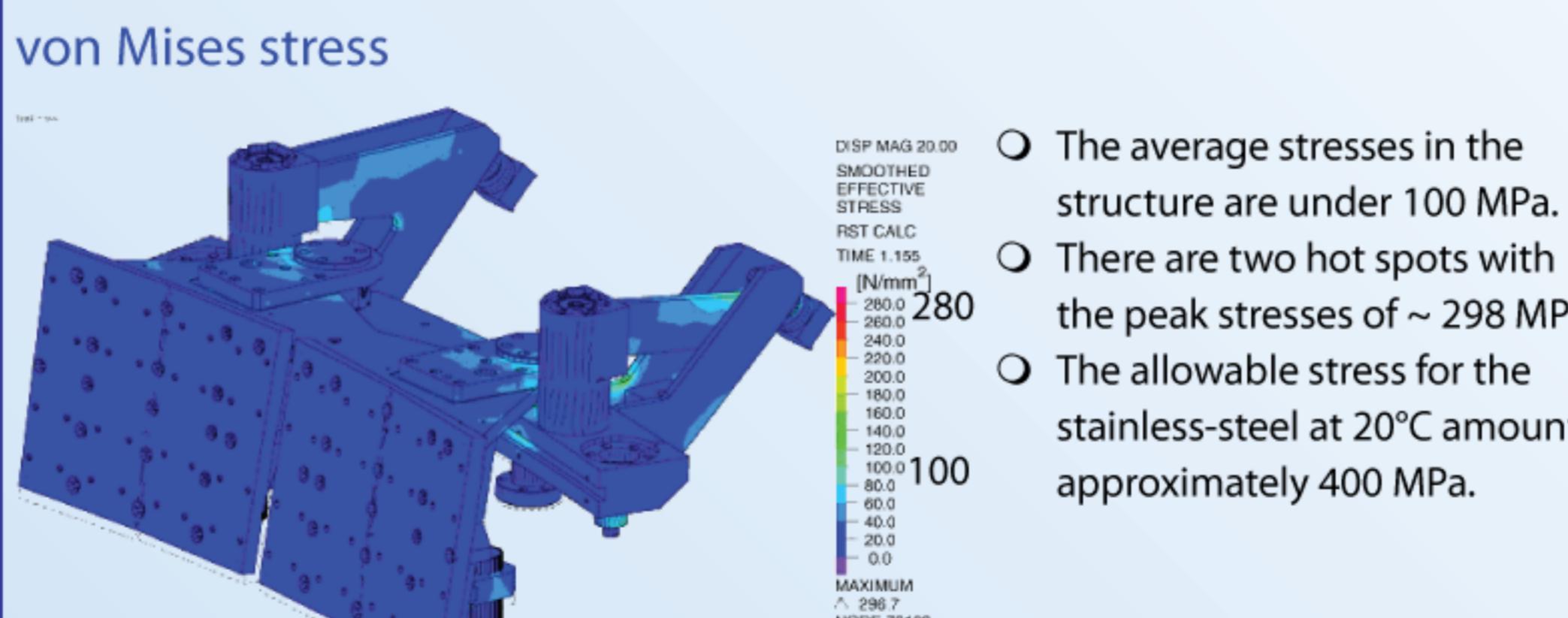
Target clamping forces

- Preload force of 2x4 stacks of disc springs: 4.8 kN
- Rupture force of 2x4 M5 fastening screws: about 68 kN

Displacement



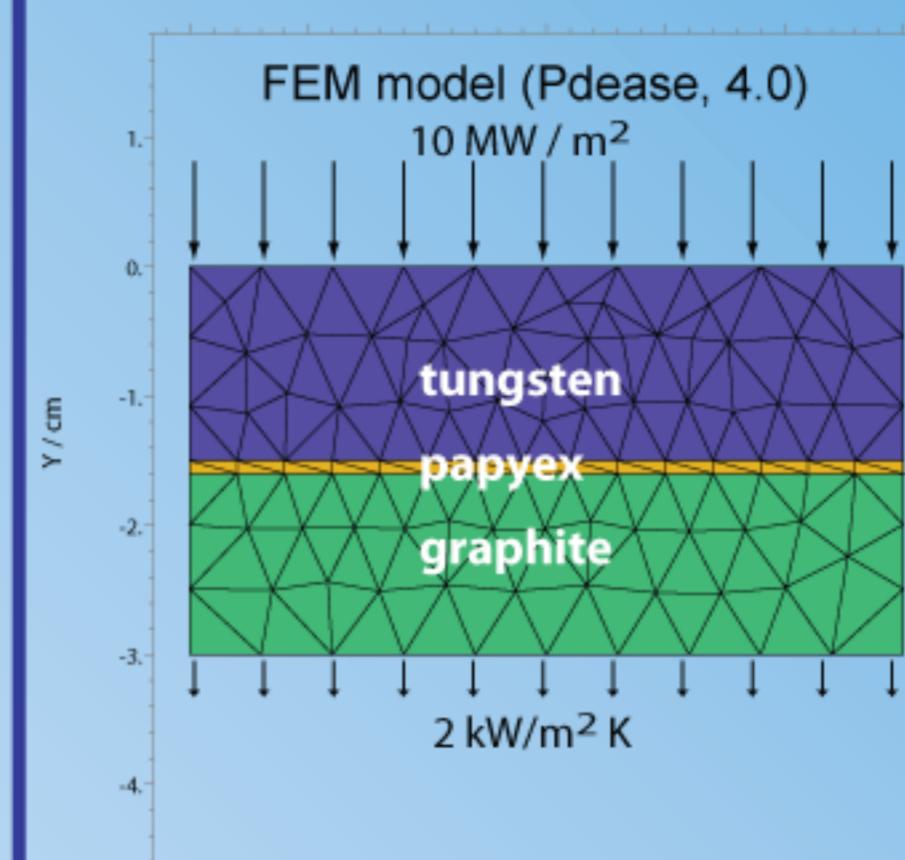
von Mises stress



- The average stresses in the structure are under 100 MPa.
- There are two hot spots with the peak stresses of ~ 298 MPa.
- The allowable stress for the stainless-steel at 20°C amounts approximately 400 MPa.
- The maximal deflection of the structure amounts only ~ 0.1mm
- The disturbance of the structure caused by the load impulse, take place ~ 0.7 s only.
- The value of the dynamic maximal deflection is far below the values gained by the static analysis.
- The design values for the divertor plate and the related support structure gained in the static analyses covered entirely all other load scenarios.

Thermal properties

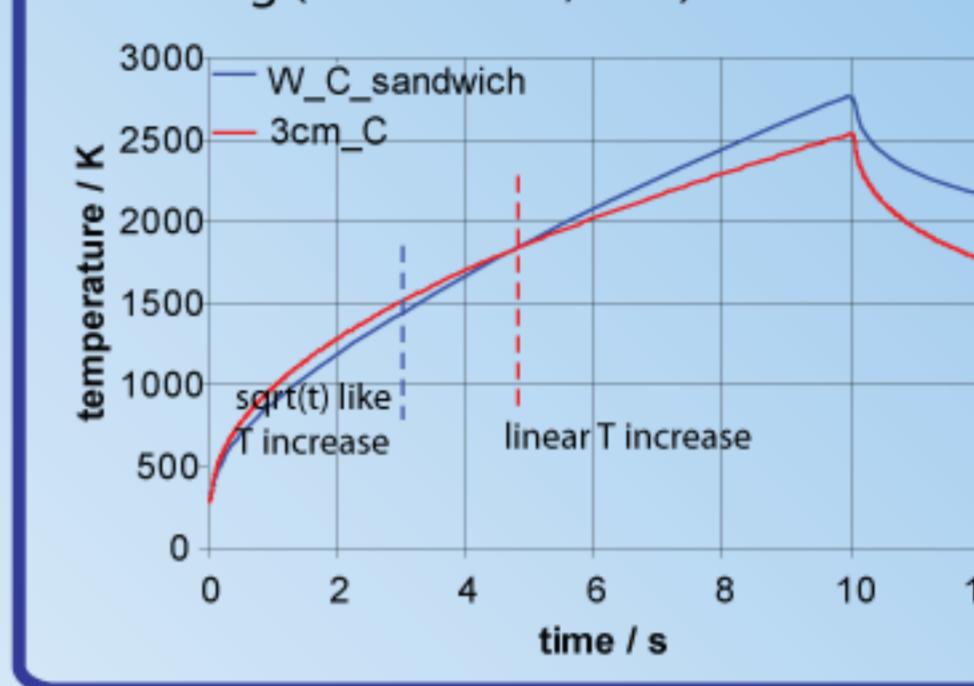
Heating and cooling - 2D FEM



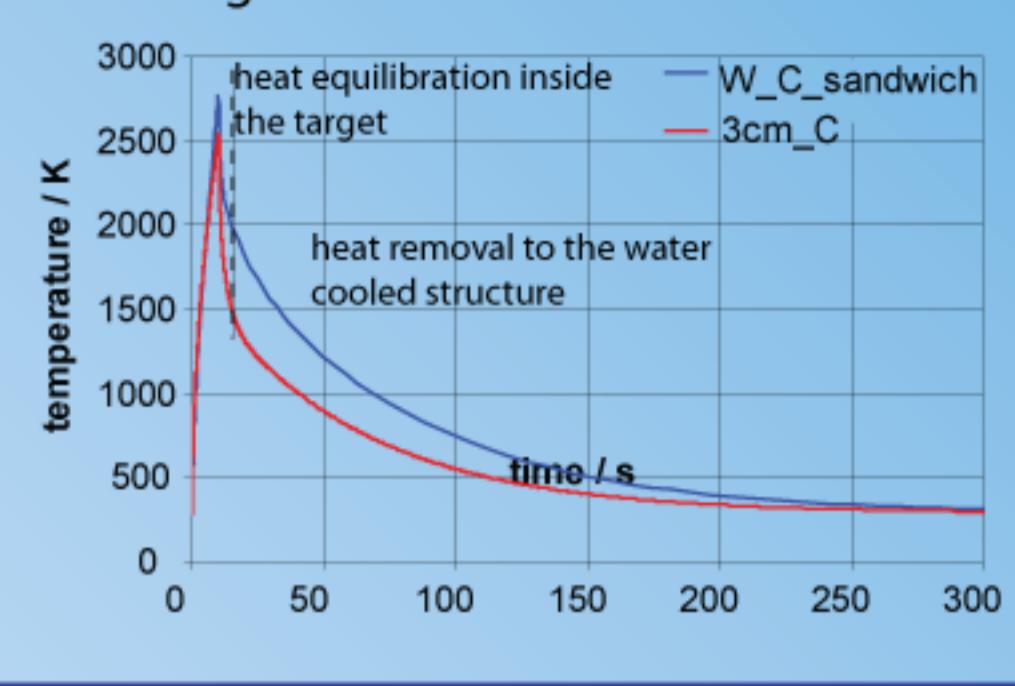
- The heat transmission between tungsten and graphite is low.
- No energy flow into graphite during the discharge.
- Heat resistance of the system is dominated by papayex (1mm) and the cooling (2 mm Papayex).
- Heat transmission coefficients:
 - 2 mm Papayex clamping: 2 kW/m² K
 - 15 mm graphite: 5 kW/m² K
- T increase becomes linear at about 3 s for the tungsten target.
- The maximum temperature is about 300 K higher compared to the graphite target.
- The T decay is slower
- The start temperature is reached at 5 min.

Temperature evolution

Heating (10 MW/m², 10 s)



Cooling



High heat flux tests in GLADIS

Garching Large Divertor Sample Test Facility - GLADIS

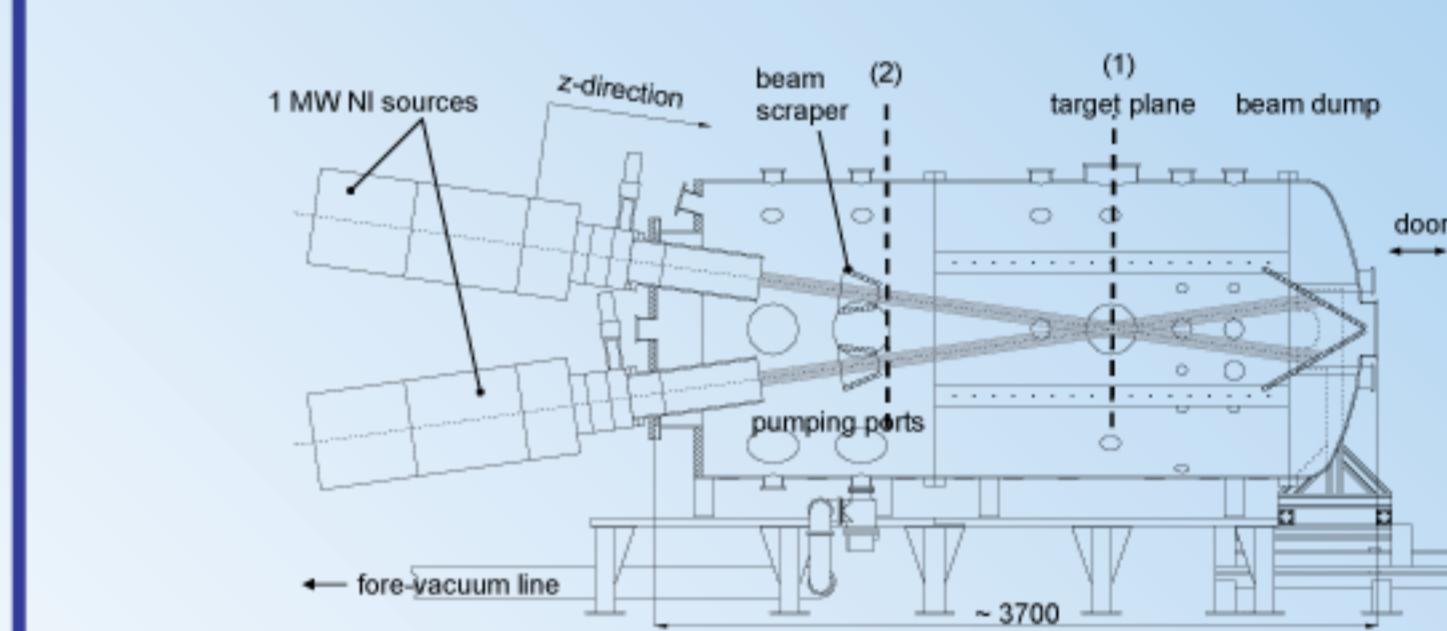


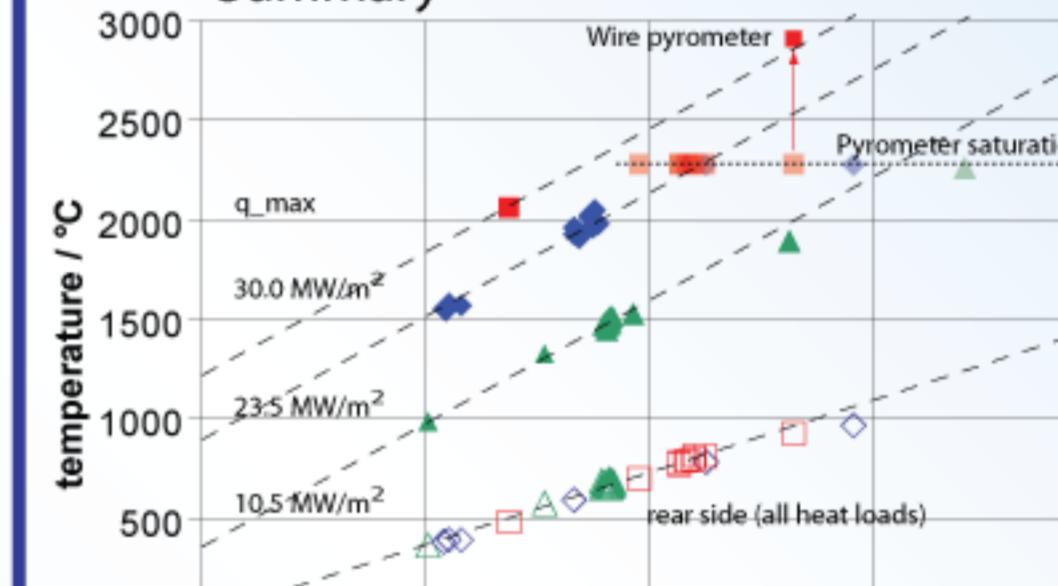
Table I
Technical characteristics of the IPP high heat flux test facility
Maximum power of the beam 1100 (2200) kW
Maximum repetition voltage 35 kV
Maximum current 20 A
Heat flux density 2-65 MW/m²
Heat flux density - time 1-15 s (9 s)
Number of cycles 100
Base plate material density 7.8 g/cm³
Base plate material density 200-700 mm
Horizontal/vertical 200-700 mm
Maximum cooling water temperature 20-30 °C
Maximum cooling water pressure 21-25 bar
Water flow rate 0.5-8 L/s (91 s)
Breakout mark values with the potential for upgrading

Aims of extensive target tests:

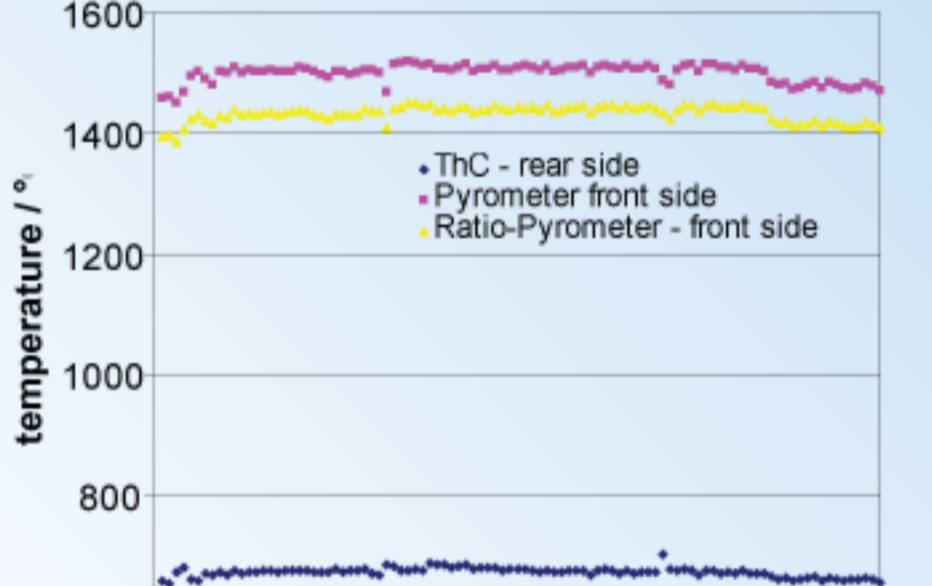
- Cyclic loading with 10.5 MW/m² for 3.5 s and 100 cycles ... representing the 'typical' ASDEX Upgrade divertor load for high power (10-15 MW) discharges.
- Find out the damage level and the consequences of a target overload by increasing the pulse length and maximum heat flux.
- Increase the surface temperature above the melting limit for carbidized tungsten at about 2800°C.

Results

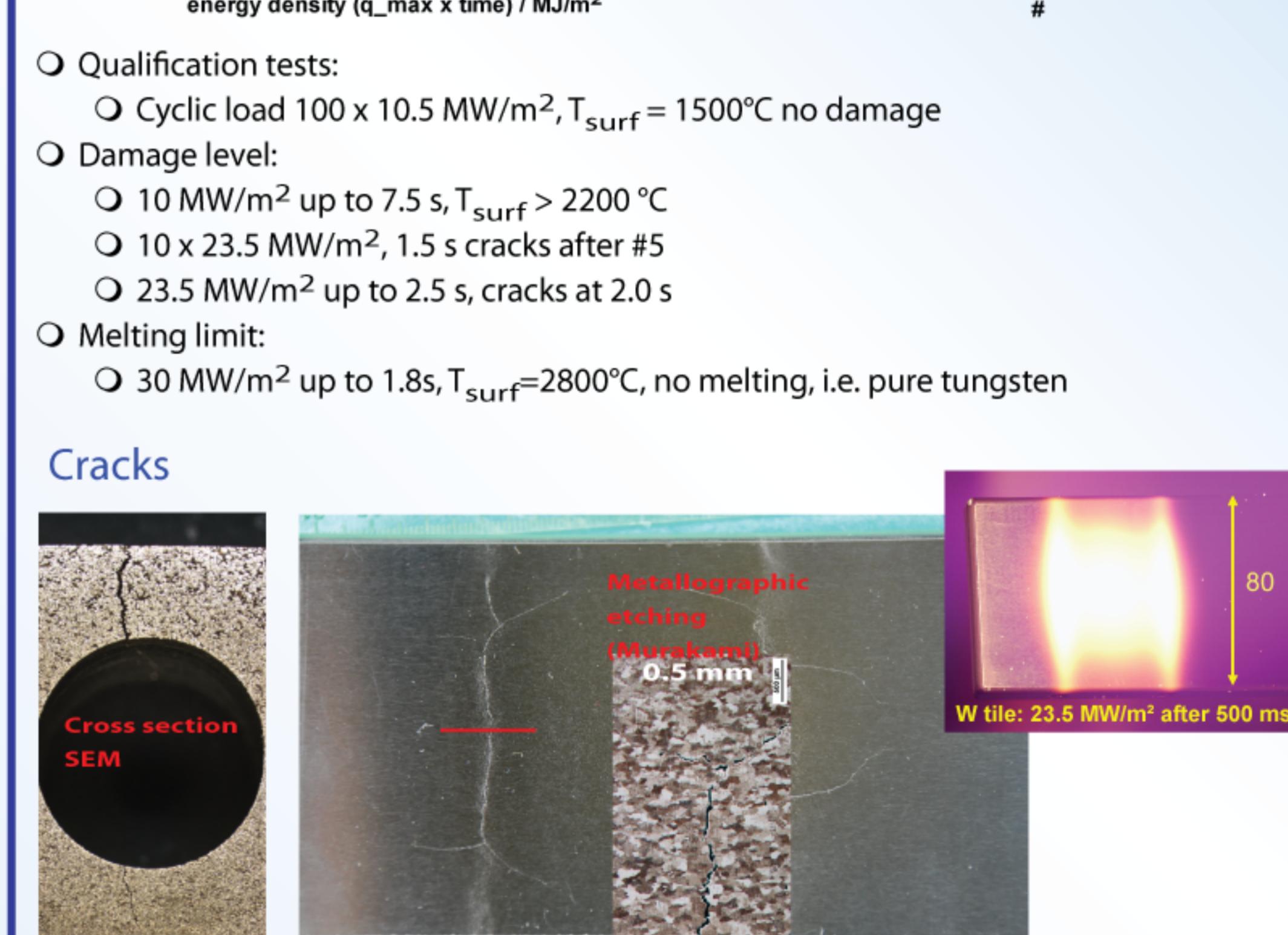
Summary



Target qualification



Cracks



Summary

- Solid tungsten divertor compatible to the existing support structure.
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